

Fig. P3.76 and P3.77

**3.66** Using an allowable shearing stress of 50 MPa, design a solid steel shaft to transmit 15 kW at a frequency of (a) 30 Hz, (b) 60 Hz.

**3.76** The two solid shafts and gears shown are used to transmit 16 hp from the motor at A operating at a speed of 1260 rpm, to a machine tool at D. Knowing that the maximum allowable shearing stress is 8 ksi, determine the required diameter (a) of shaft AB, (b) of shaft CD.

**3.83** A 1.5-m-long tubular steel shaft of 38-mm outer diameter  $d_1$  is to be made of a steel for which  $\tau_{all} = 65$  MPa and  $G = 77.2$  GPa. Knowing that the angle of twist must not exceed  $4^\circ$  when the shaft is subjected to a torque of  $600$  N · m, determine the largest inner diameter  $d_2$  that can be specified in the design.

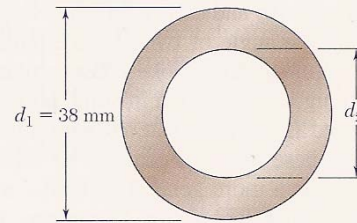


Fig. P3.82 and P3.83

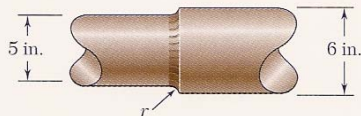


Fig. P3.84 and P3.85

**3.84** The stepped shaft shown rotates at 450 rpm. Knowing that  $r = 0.2$  in., determine the largest torque  $T$  that can be transmitted without exceeding an allowable shearing stress of 7500 psi.

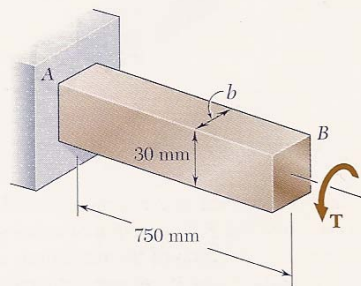


Fig. P3.125 and P3.126

**3.126** The torque  $T$  causes a rotation of  $0.6^\circ$  at end B of the aluminum bar shown. Knowing that  $b = 15$  mm and  $G = 26$  GPa, determine the maximum shearing stress in the bar.

**3.136** A 3-m-long steel angle has an L203 × 152 × 12.7 cross section. From Appendix C we find that the thickness of the section is 12.7 mm and that its area is  $4350$  mm<sup>2</sup>. Knowing that  $\tau_{all} = 50$  MPa and that  $G = 77.2$  GPa, and ignoring the effect of stress concentration, determine (a) the largest torque  $T$  that can be applied, (b) the corresponding angle of twist.

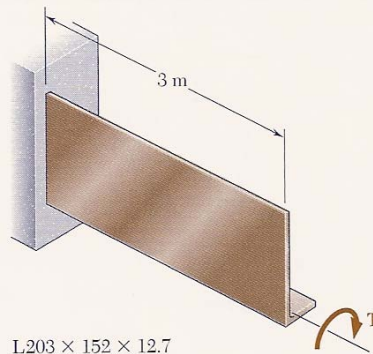


Fig. P3.136